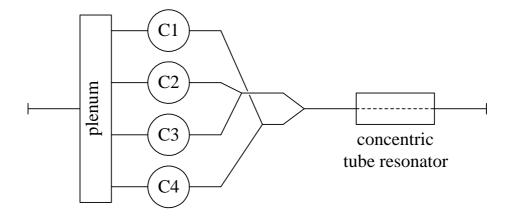
Simulation of a Concentric-Tube Resonator

P. Bartsch, B. Bachner, A. Borzì, and H.A. Schuemie

Objective: Accurate modeling and fast simulation of silencers is important for the purpose of their design aiming at reducing engine exhaust noise.

Applications: Manufacturing of exhaust systems, tube-resonators, mufflers.



A four cylinder engine model with silencer.

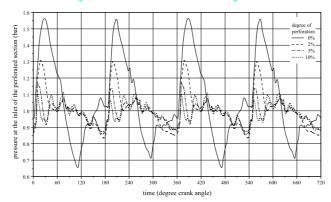
Modeling: At higher excitation frequencies we use a modification of the St. Venant equation for the mass flow rate through the holes of the silencer:

$$\dot{m} = c_d \, p_0 \sqrt{\frac{2}{RT_0}} \sqrt{\frac{\kappa}{\kappa - 1} \left[\left(\frac{p}{p_0} \right)^{\frac{2}{\kappa}} - \left(\frac{p}{p_0} \right)^{\frac{\kappa + 1}{\kappa}} \right] - \left(\frac{p}{p_0} \right)^{\frac{2}{\kappa}} \frac{1}{RT_0} \frac{\partial w}{\partial t} l}, \tag{1}$$

Used to determine the source terms of the Euler Equations of Gas Dynamics for the flow in the silencer:

$$\frac{\partial}{\partial t} \begin{pmatrix} \rho \\ \rho u \\ e \end{pmatrix} + \frac{\partial}{\partial x} \begin{pmatrix} \rho u \\ \rho u^2 + p \\ u(e+p) \end{pmatrix} = -\frac{1}{S} \frac{dS}{dx} \begin{pmatrix} \rho u \\ \rho u^2 \\ u(e+p) \end{pmatrix} - \frac{1}{S} \begin{pmatrix} Q_m \\ 0 \\ Q_e \end{pmatrix}, \tag{2}$$

A result: The influence of the perforation on the pressure curve



Pressure at the inlet of the perforated section versus time, expressed in degree crank angle.